

**Amendments to the Specification**

Please replace paragraph [0012] of the specification with the following:

[0012] A spring-pressure brake 1 according to Figures 1 and 2 is hydraulically operated in a manner which will be described in greater detail hereinafter. The spring-pressure brake is used to brake a hydromotor not illustrated in detail. The spring-pressure brake 1 has for this purpose a brake shaft 2 which is rotatably movably supported about an axis of rotation D. The brake shaft 2 can be connected at the front left end in a manner not illustrated in detail to a corresponding drive for the hydromotor in order to achieve a power transfer from the spring-pressure brake 1 onto the hydromotor. The brake shaft 2 is supported by means of a roller-bearing arrangement 15 in a brake housing, which is essentially defined by an outside body 6 designed as a one-piece hollow body. The outside body 6 is designed as a one-piece metal casting and has an essentially hollow cylindrical configuration. The outside body 6 is designed essentially rotationally symmetrically with respect to the axis of rotation D but for the flange sections 16, 17, which will be described in greater detail hereinafter. The inside of the outside body 6 is open toward both axial ends. The braking shaft 2 extends coaxially into the inside of the outside body 6. Axially following the described roller-bearing arrangement 15 for the rotational support of the brake shaft 2, the brake shaft 2 has a brake hub 3 which has an external tooth system for the axial guiding of disks of a disk package 4 serving as a disk-brake arrangement. Disks are positioned between each of the disks rotationally lockingly connected to the brake hub 3, which disks are held fixed against rotation, however, are axially movable on the inner circumference of the outside body 6. The axial mobility of the disks of the disk package 4 is defined on a side facing the front left end directly adjacent

to the roller-bearing arrangement 15 by a brake ring 21 which is axially secured in an annular groove 22 in the inner circumference of the outside body 6, ~~which annular groove is not identified in detail~~. The disk package 4 can on the opposite axial end thereof be axially loaded by a brake-piston arrangement 5, in the present case by an annular brake piston, for a braking operation. The brake-piston arrangement 5 is axially movably supported on the inner circumference of the outside body 6.

IN THE ABSTRACT OF THE DISCLOSURE

Attached hereto is a replacement Abstract.